

EQUIPOS E INSTALACIONES INDUSTRIALES

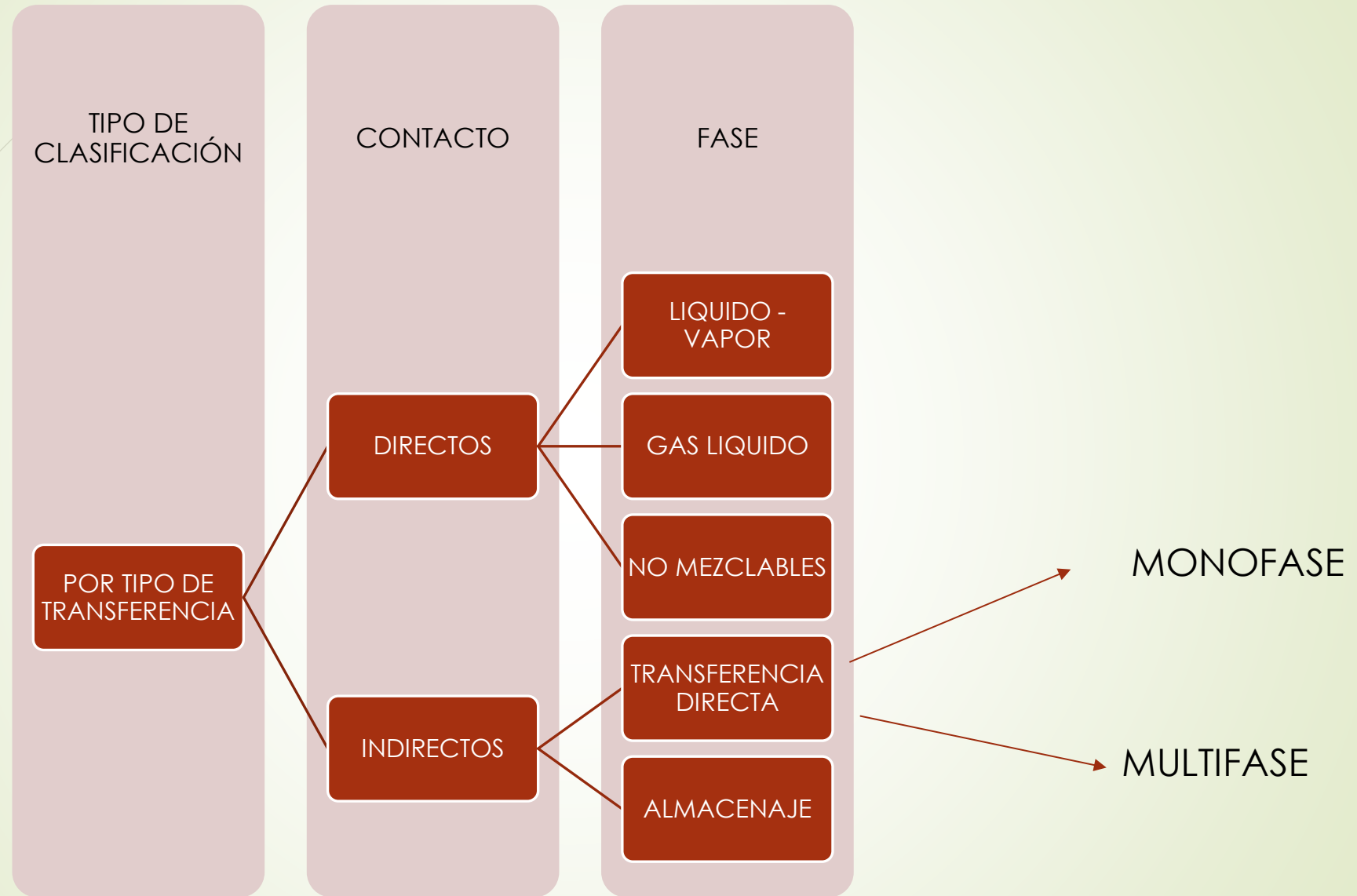
PROFESOR: ING. JORGE NOZICA

PROFESOR: ING. HÉCTOR PÉREZ

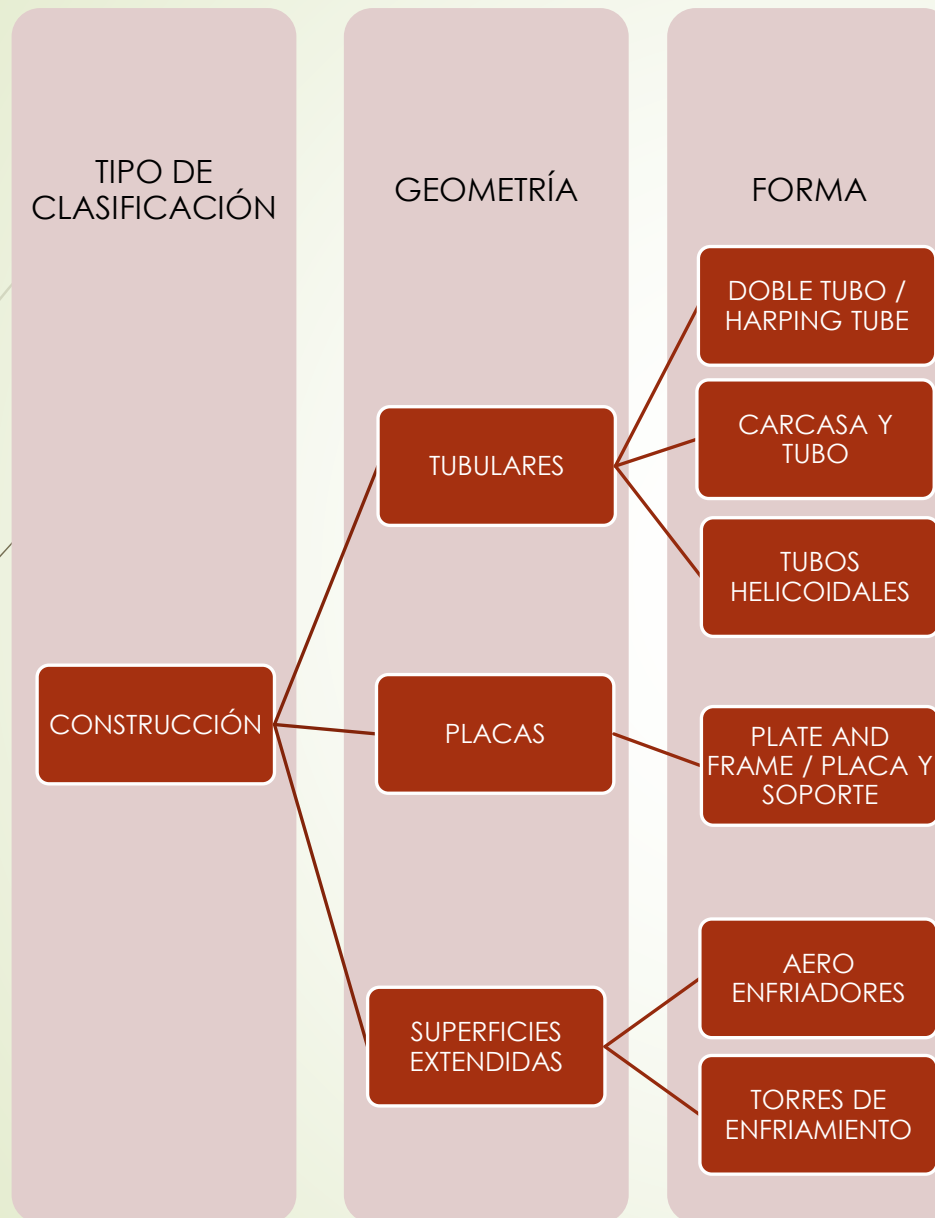
PROFESOR: ING. LETICIA SIMONCINI

INTERCAMBIADORES DE CALOR

INTERCAMBIADORES DE CALOR CLASIFICACIÓN GENERAL

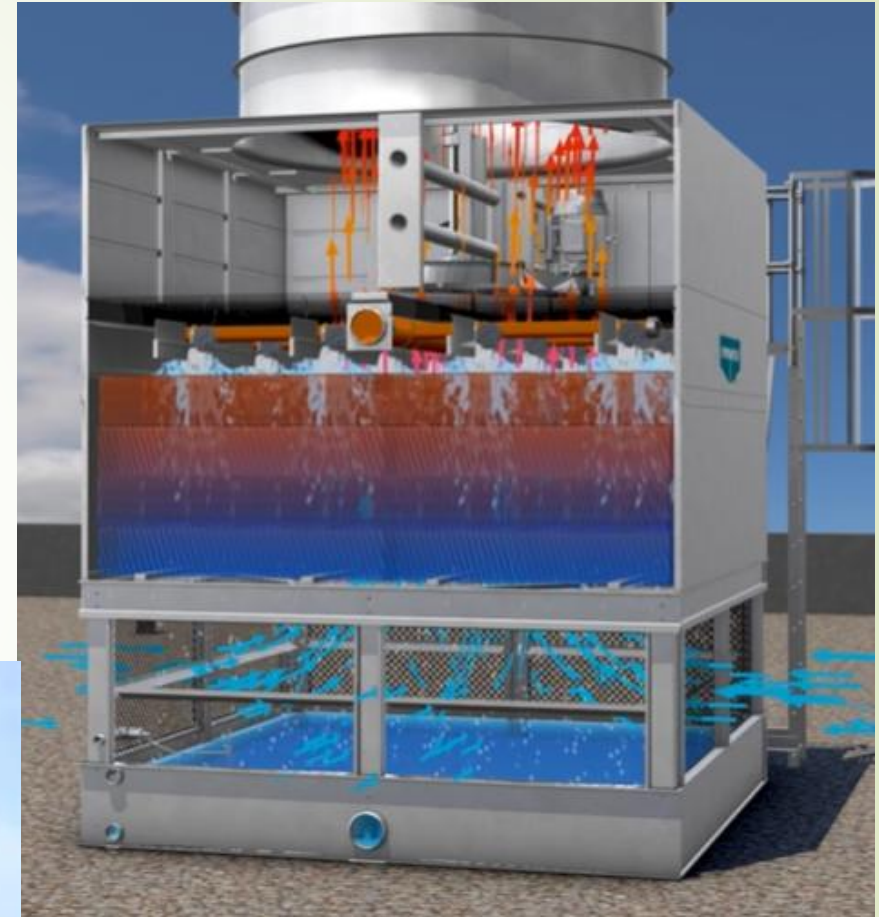


CLASIFICACIÓN DE FABRICACIÓN



CONTACTO DIRECTO

■ TORRE DE ENFRIAMIENTO – COOLING TOWER



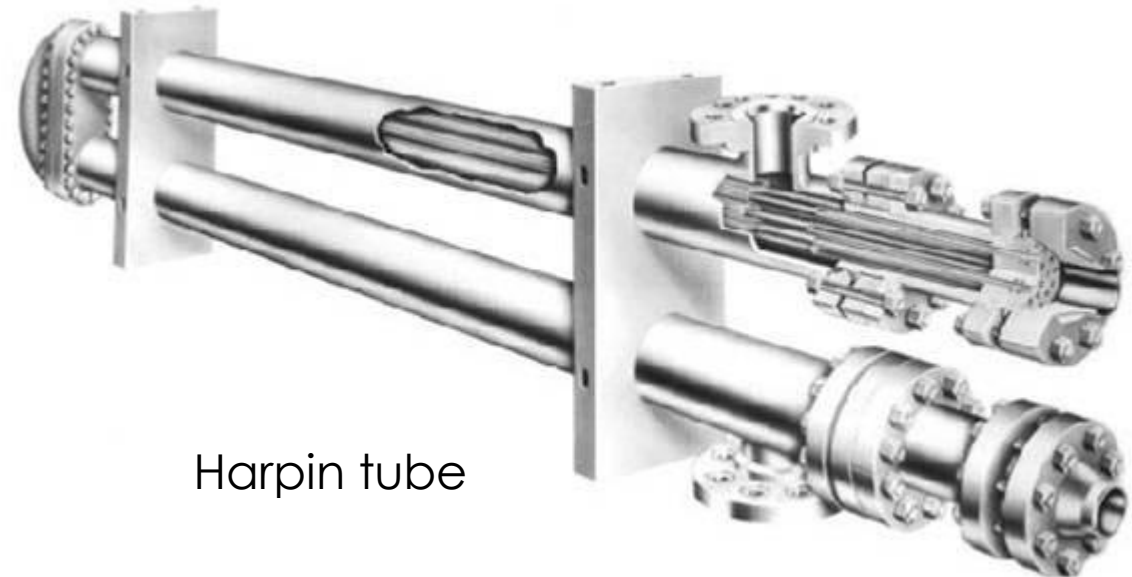
CONTACTO INDIRECTO

▀ AERO ENFRIADORES



INTERCAMBIADORES TUBO EN TUBO Y HARPIN TUBE

Doble ánulo



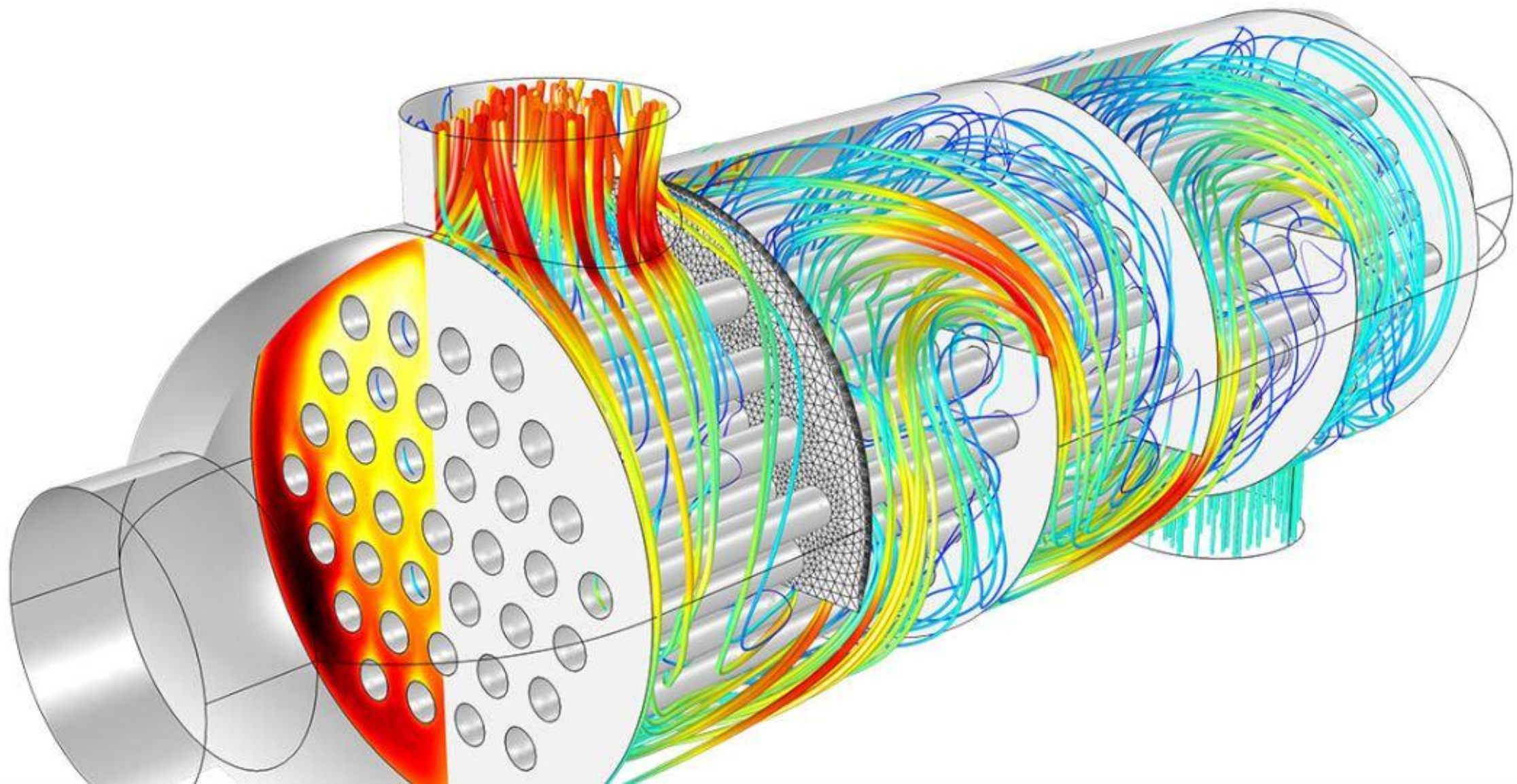
Harpin tube

INTERCAMBIADORES DE CALOR

- INTERCAMBIADOR DE CALOR
CARCASA Y TUBO (C&T)



INTERCAMBIADOR C&T



DISEÑO TÉRMICO Y DISEÑO MECÁNICO

DISEÑO TÉRMICO

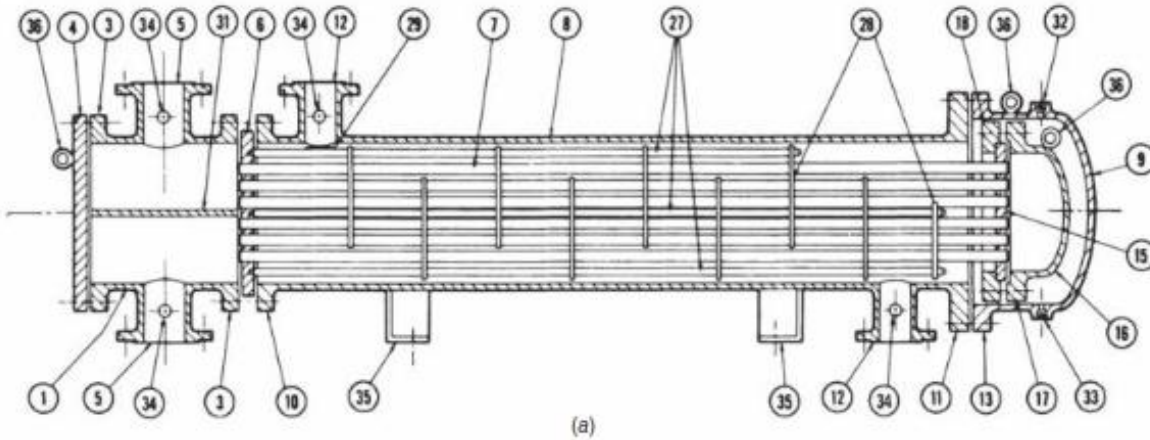
- Coeficientes peliculares ($h_{io} - h_o$) – función de n^a adimensionales
- Coeficiente global U (tabla 11-3 / 11-4 / 11-5 Perry 's Chemical Engineers Handbook)
- Resistencias adicionales (fouling factors)
- Diferencia de temperatura efectiva (ΔT_{ln})
- Área de intercambio
- Carga térmica $Q = U \times A \times DT$ (kcal/h / kw)
- Pre dimensionamiento geométrico de tubos y carcasa
- Pérdida de carga
- Condiciones de operación: Temperaturas, presiones, tipo de fluidos

DISEÑO TÉRMICO Y DISEÑO MECÁNICO

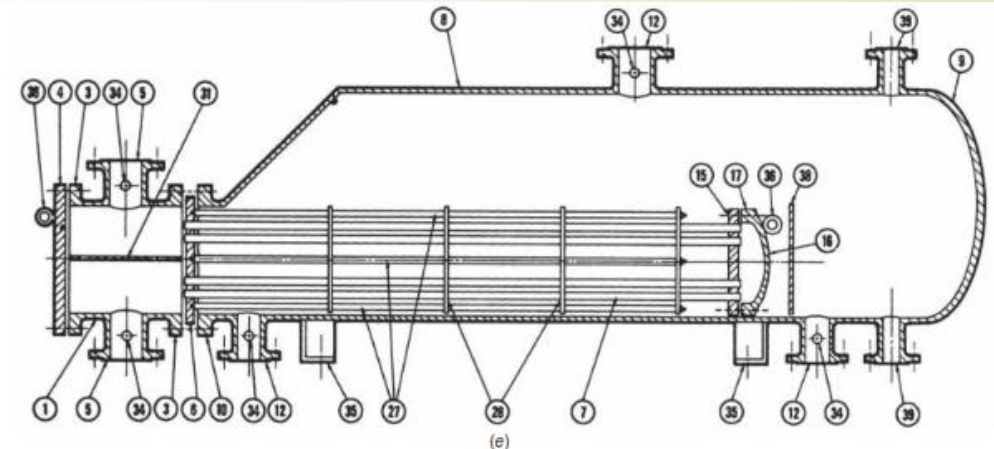
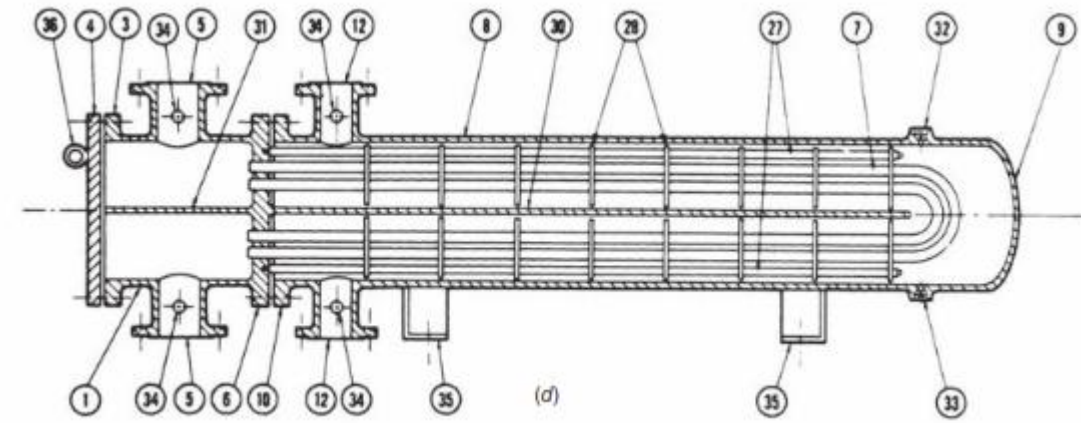
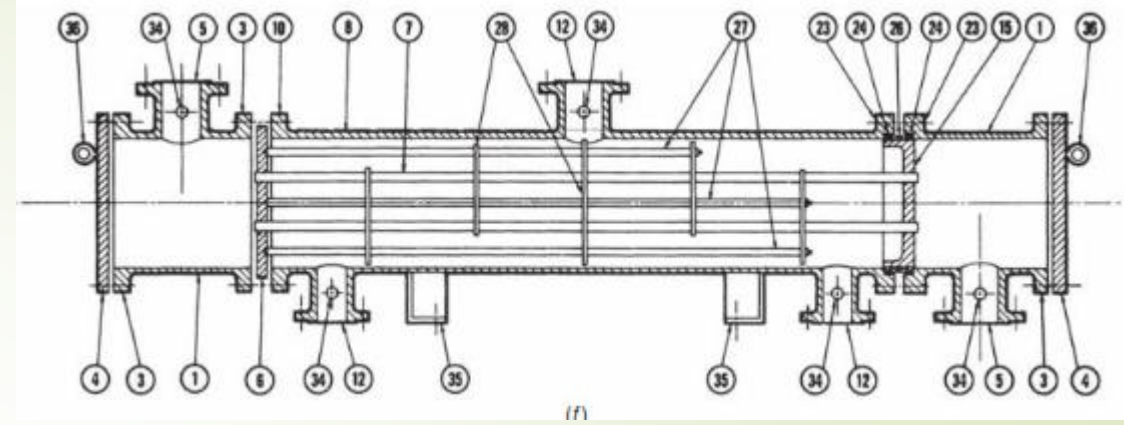
➤ DISEÑO MECÁNICO

- Espesores de carcasa
- Verificación de tubos (tabla 11-12 Perry's Chemical Engineers Handbook)
- Tube sheet (placa porta tubos)
- Baffles
- Bridas
- Pernos
- Aislación

NORMAS TEMA



- | | |
|---|--|
| 1. Stationary Head—Channel | 20. Slip-on Backing Flange |
| 2. Stationary Head—Bonnet | 21. Floating Head Cover—External |
| 3. Stationary Head Flange—Channel or Bonnet | 22. Floating Tubesheet Skirt |
| 4. Channel Cover | 23. Packing Box Flange |
| 5. Stationary Head Nozzle | 24. Packing |
| 6. Stationary Tubesheet | 25. Packing Gland |
| 7. Tubes | 26. Lantern Ring |
| 8. Shell | 27. Tie Rods and Spacers |
| 9. Shell Cover | 28. Transverse Baffles or Support Plates |
| 10. Shell Flange—Stationary Head End | 29. Impingement Plate |
| 11. Shell Flange—Rear Head End | 30. Longitudinal Baffle |
| 12. Shell Nozzle | 31. Pass Partition |
| 13. Shell Cover Flange | 32. Vent Connection |
| 14. Expansion Joint | 33. Drain Connection |
| 15. Floating Tubesheet | 34. Instrument Connection |
| 16. Floating Head Cover | 35. Support Saddle |
| 17. Floating Head Flange | 36. Lifting Lug |
| 18. Floating Head Backing Device | 37. Support Bracket |
| 19. Split Shear Ring | 38. Weir |
| | 39. Liquid Level Connection |



Alcance y Nomenclatura de Norma TEMA

- DIÁMETRO DE CARCASA 100" (2540mm)
- Pd < 3000 psi (210 bar)
- Espesor de carcasa 3" (76mm)

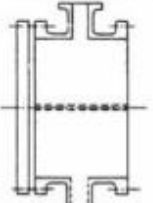
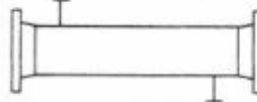
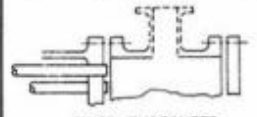

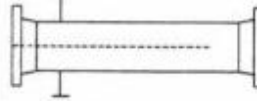
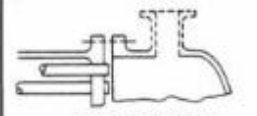
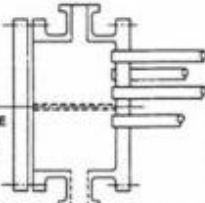
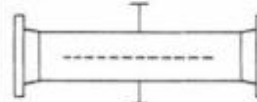

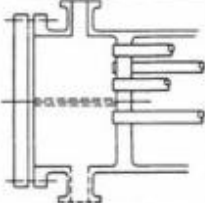
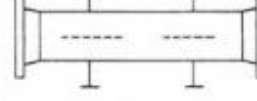
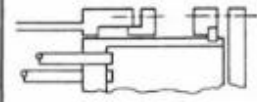
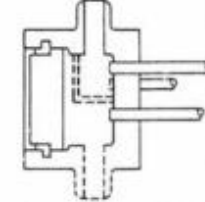

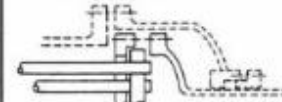

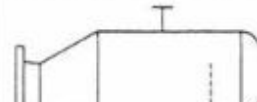
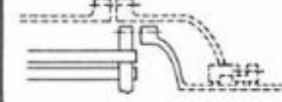
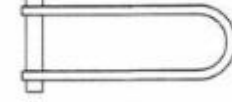
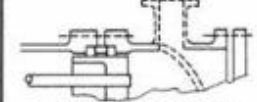
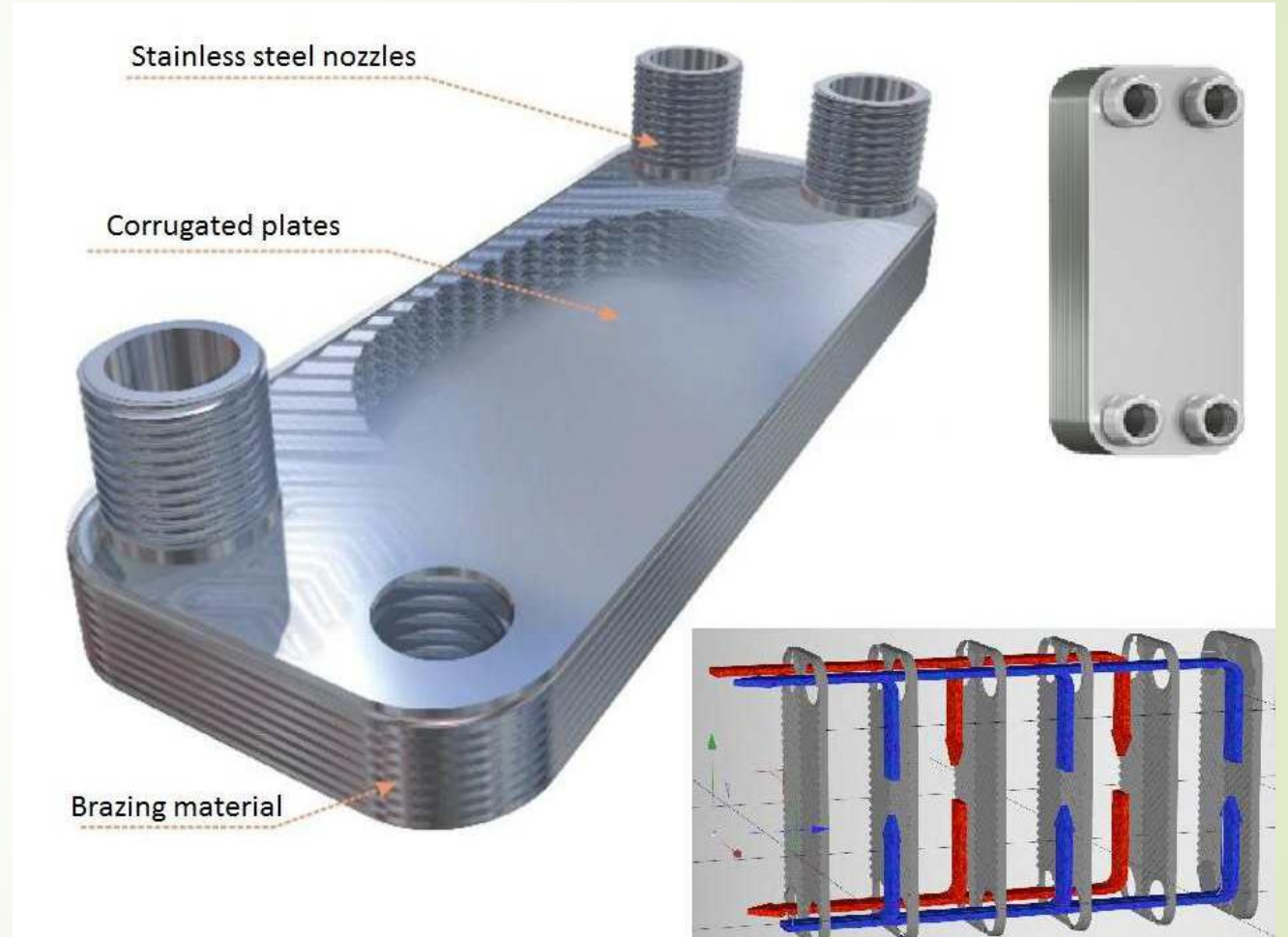
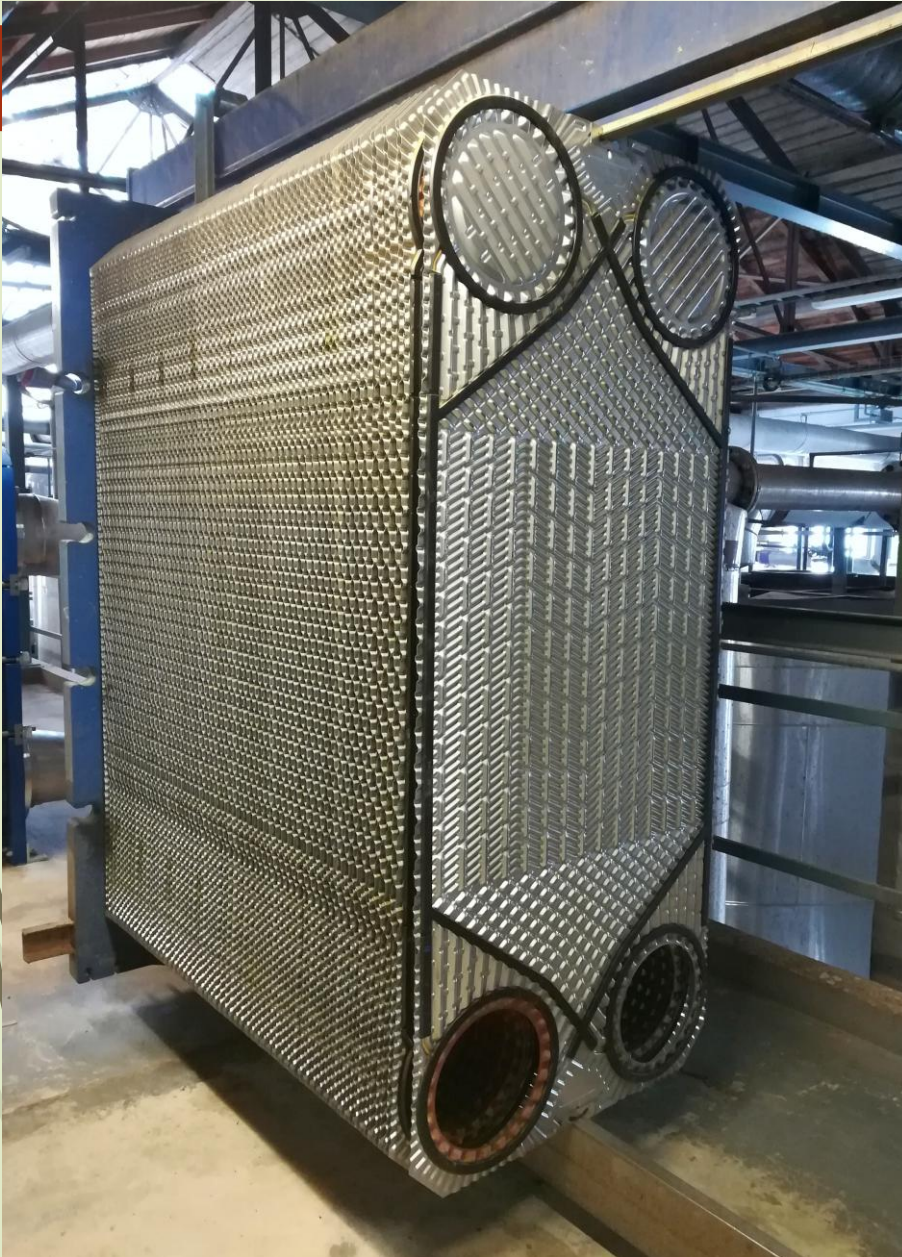
FRONT END STATIONARY HEAD TYPES		SHELL TYPES		REAR END HEAD TYPES	
A		E		L	
	CHANNEL AND REMOVABLE COVER		ONE PASS SHELL		FIXED TUBESHEET LIKE "A" STATIONARY HEAD
B		F		M	
	BONNET (INTEGRAL COVER)		TWO PASS SHELL WITH LONGITUDINAL BAFFLE		FIXED TUBESHEET LIKE "B" STATIONARY HEAD
C		G		N	
	REMOVABLE TUBE BUNDLE ONLY		SPLIT FLOW		FIXED TUBESHEET LIKE "N" STATIONARY HEAD
N		H		P	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER		DOUBLE SPLIT FLOW		OUTSIDE PACKED FLOATING HEAD
D		J		S	
	CHANNEL INTEGRAL WITH TUBE- SHEET AND REMOVABLE COVER		DIVIDED FLOW		FLOATING HEAD WITH BACKING DEVICE
X		K		T	
	SPECIAL HIGH PRESSURE CLOSURE		KETTLE TYPE REBOILER		PULL THROUGH FLOATING HEAD
		U		W	
			CROSS FLOW		EXTERNALLY SEALED FLOATING TUBESHEET

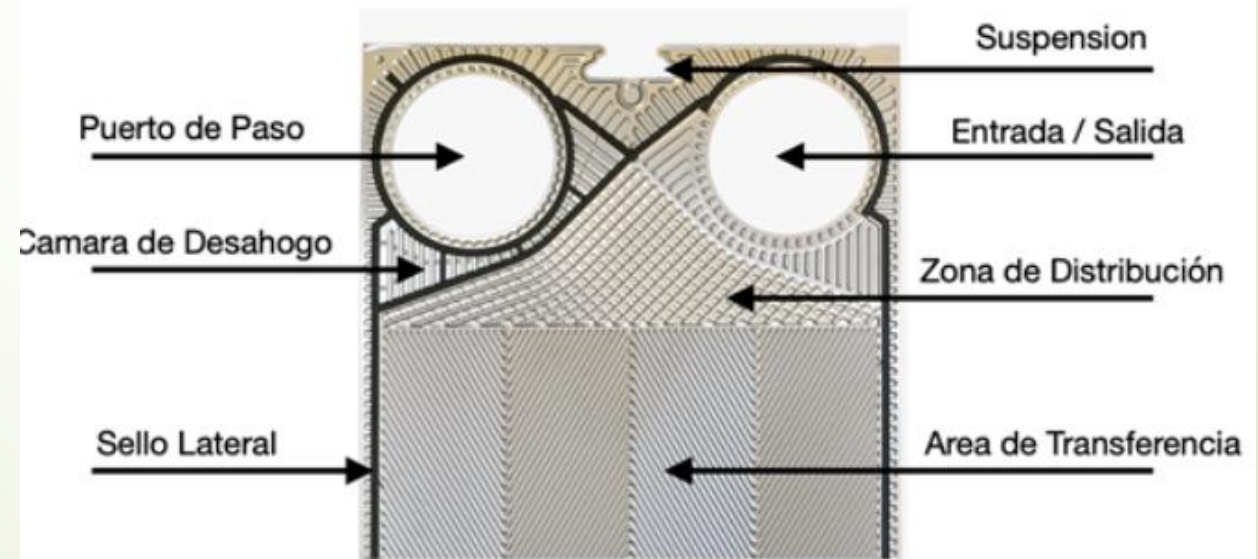
FIG. 11-35 TEMA-type designations for shell-and-tube heat exchangers. (Standards of Tubular Exchanger Manufacturers Association, 6th ed., 1978.)

INTERCAMBIADORES A PLACAS

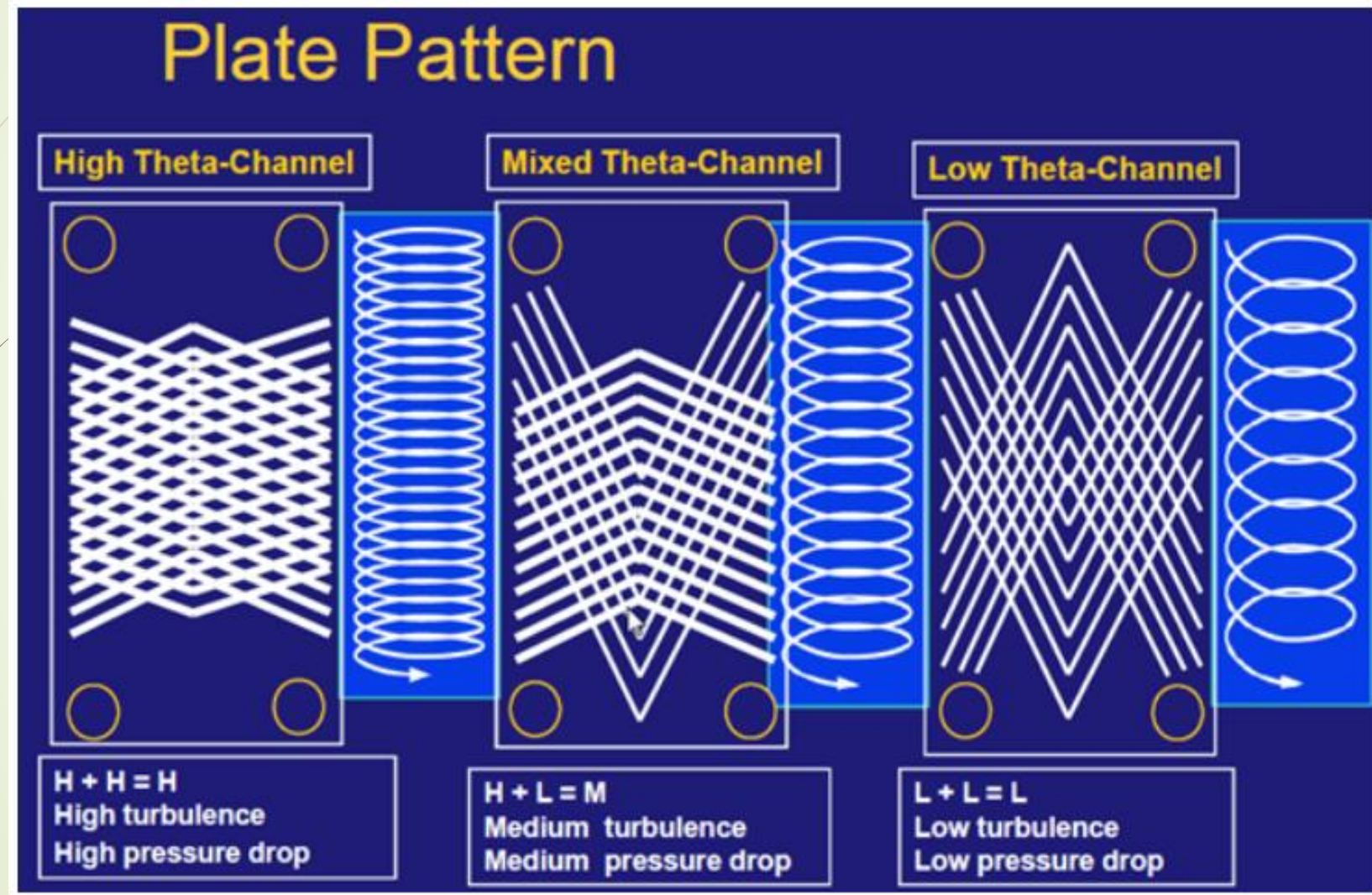


<https://www.youtube.com/watch?v=2ydy48AKd8c>

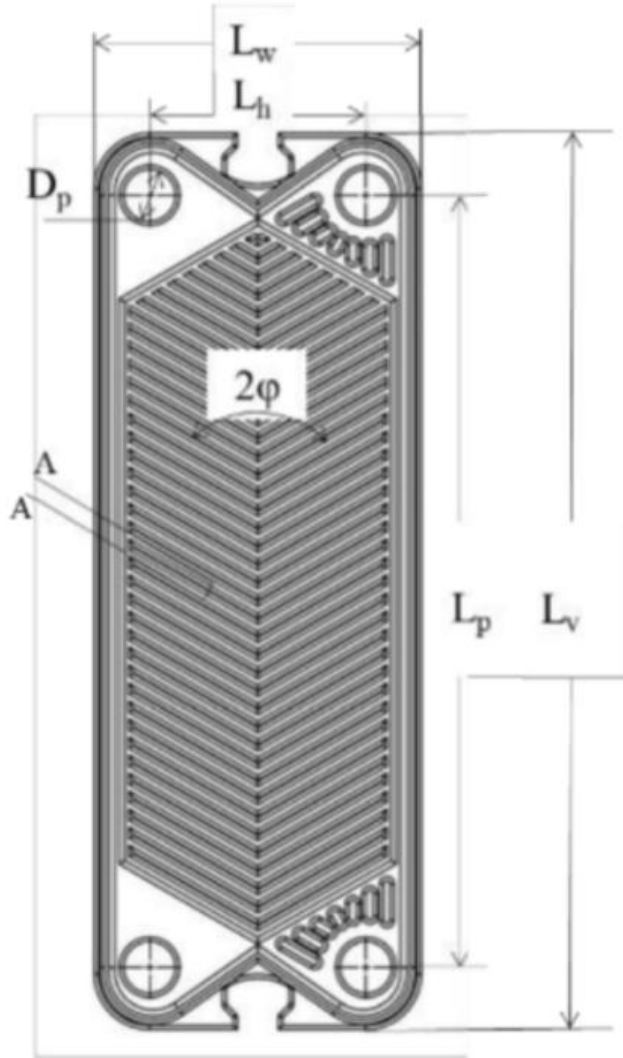
Principios



Patrones de flujo



Especificaciones



Medidas de la placa

Distancia Horizontal entre puertos (L_h)= _____ mm

Distancia Vertical entre puertos (L_p)= _____ mm

Longitud de la placa (L_v)= _____ mm

Ancho de la placa (L_w)= _____ mm

Diámetro del puerto (D_p)= _____ mm

Aero enfriadores

- Superficies extendidas finas
 - Coeficientes peliculares muy disímiles
- $hoA_{oe} \approx hiA_i$

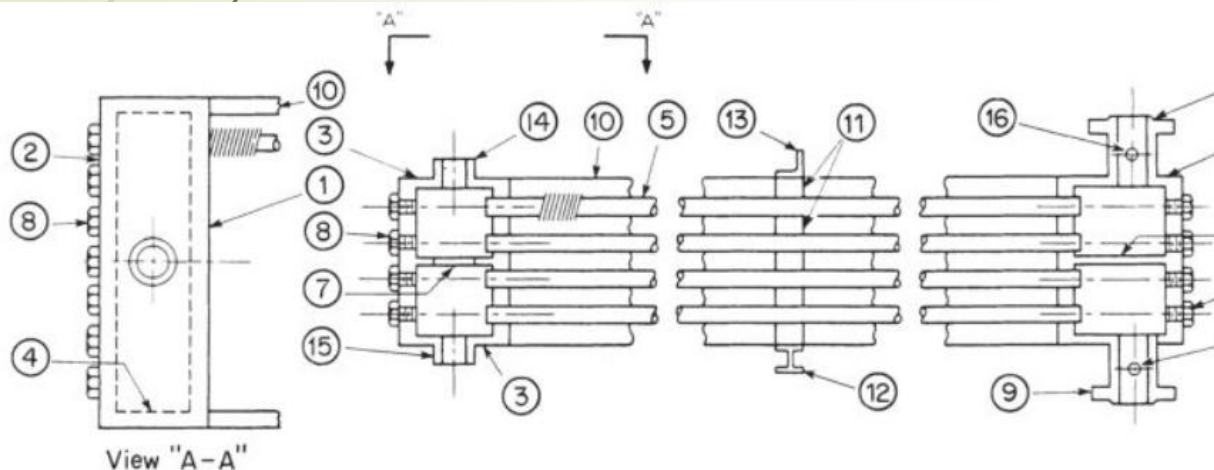
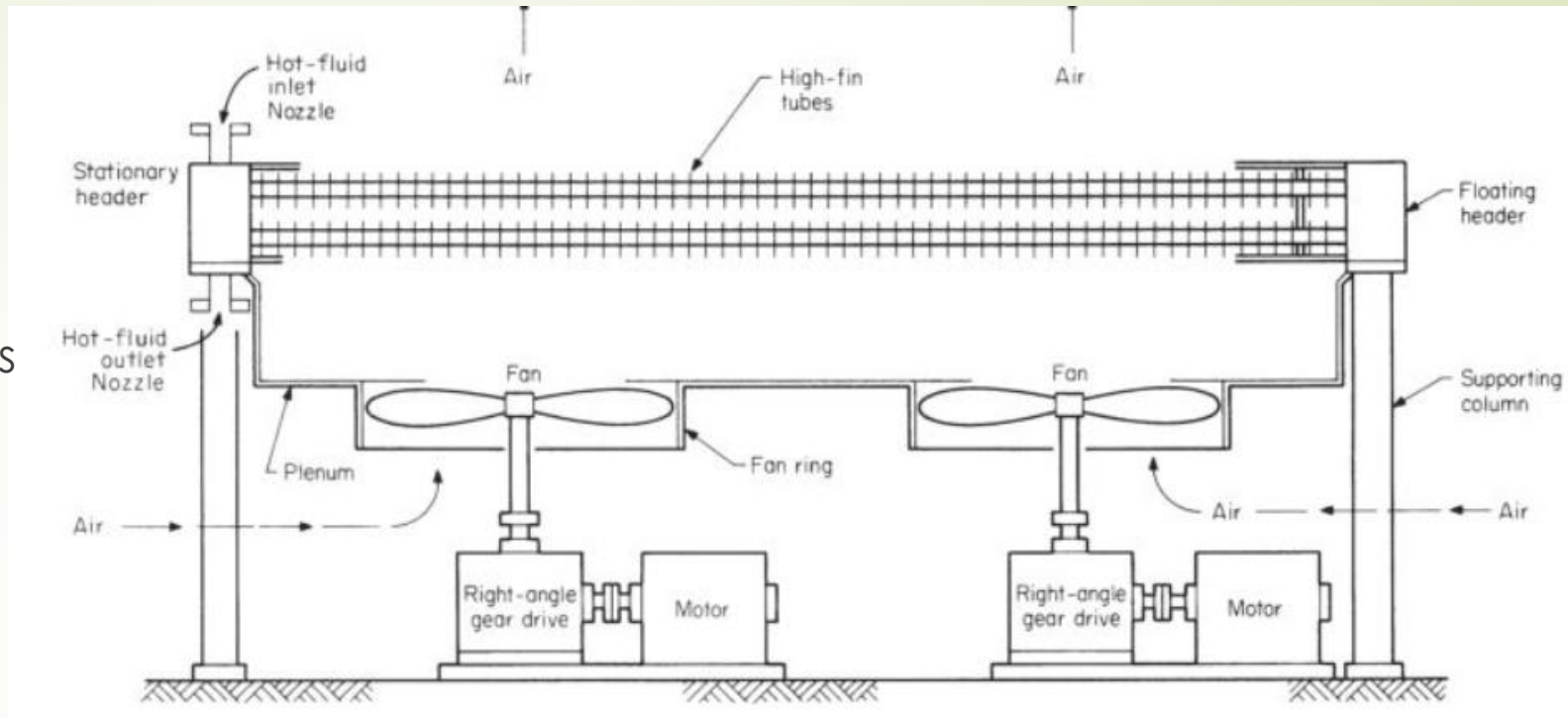


FIG. 11-44 Typical construction of a tube bundle with plug headers: (1) tube sheet; (2) plug sheet; (3) top and bottom plates; (4) end plate; (5) tube; (6) pass partition; (7) stiffener; (8) plug; (9) nozzle; (10) side frame; (11) tube spacer; (12) tube-support cross member; (13) tube keeper; (14) vent; (15) drain; (16) instrument connection. (API Standard 661.)

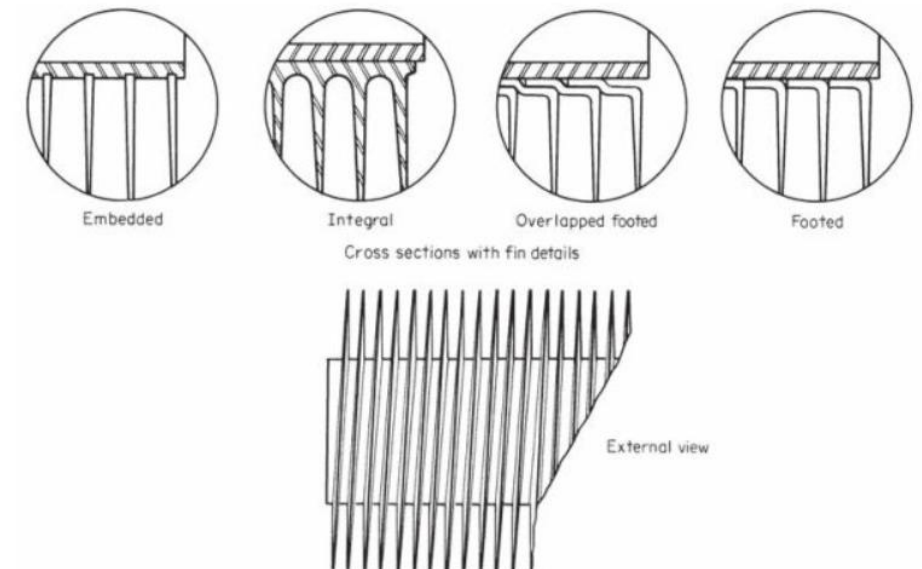


FIG. 11-45 Finned-tube construction.

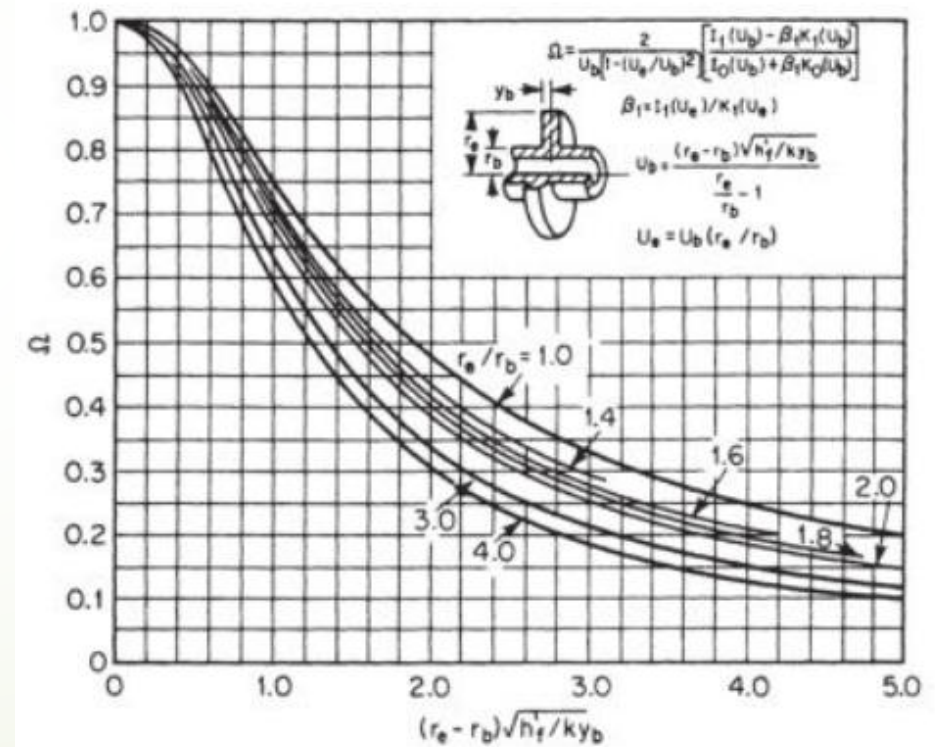
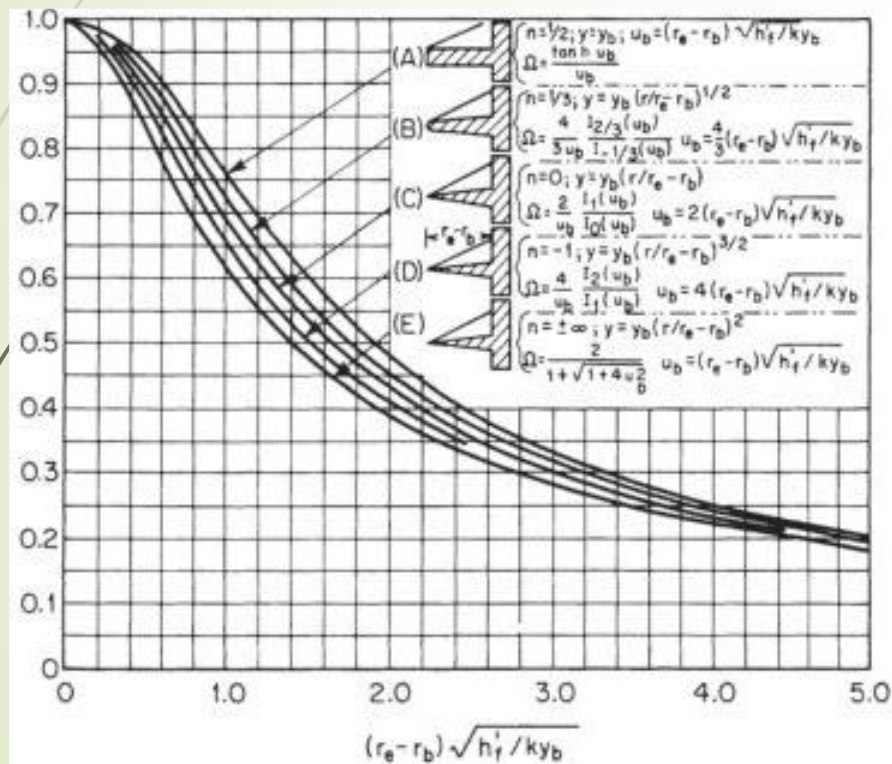
$$A_{oe} = A_{uf} + A_f \Omega$$

A_{oe}: área efectiva de transferencia total

A_{uf}: área de transferencia de tubo liso

A_f: área de aletas

Ω: eficiencia de aleta



$$Q = U \times A_{oe} \times \Delta T \text{ efectivo}$$

$$\Delta T_e = \Delta T_{ln} \times f$$

F= factor de corrección de diferencia de temperatura (ajuste a modelo de flujo)

F=0,91 1 paso

F=0,96 2 pasos

F=0,99 3 pasos (Perry's)

TABLE 11-5 Overall Coefficients for Air-Cooled Exchangers on Bare-Tube Basis

Btu/(°F · ft ² · h)			
Condensing	Coefficient	Liquid cooling	Coefficient
Ammonia	110	Engine-jacket water	125
Freon-12	70	Fuel oil	25
Gasoline	80	Light gas oil	65
Light hydrocarbons	90	Light hydrocarbons	85
Light naphtha	75	Light naphtha	70
Heavy naphtha	65	Reformer liquid	
Reformer reactor effluent	70	streams	70
Low-pressure steam	135	Residuum	15
Overhead vapors	65	Tar	7
Gas cooling	Operating pressure, lb./sq. in. gage	Pressure drop, lb./sq. in.	Coefficient
Air or flue gas	50	0.1 to 0.5	10
	100	2	20
	100	5	30
Hydrocarbon gas	35	1	35
	125	3	55
	1000	5	80
Ammonia reactor stream			85

Bare-tube external surface is 0.262 ft²/ft.

Fin-tube surface/bare-tube surface ratio is 16.9.

To convert British thermal units per hour-square foot-degrees Fahrenheit to joules per square meter-second-kelvins, multiply by 5.6783; to convert pounds-force per square inch to kilopascals, multiply by 6.895.